



Cranfield



Numerical Modeling of the Dynamics of a Tether Stack in Towed Active Debris Removal

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Agenda

- BOUNCED study
- Description of the deorbiting mission
- How do we model it ?
- Validation & Results
- Modeling capabilities



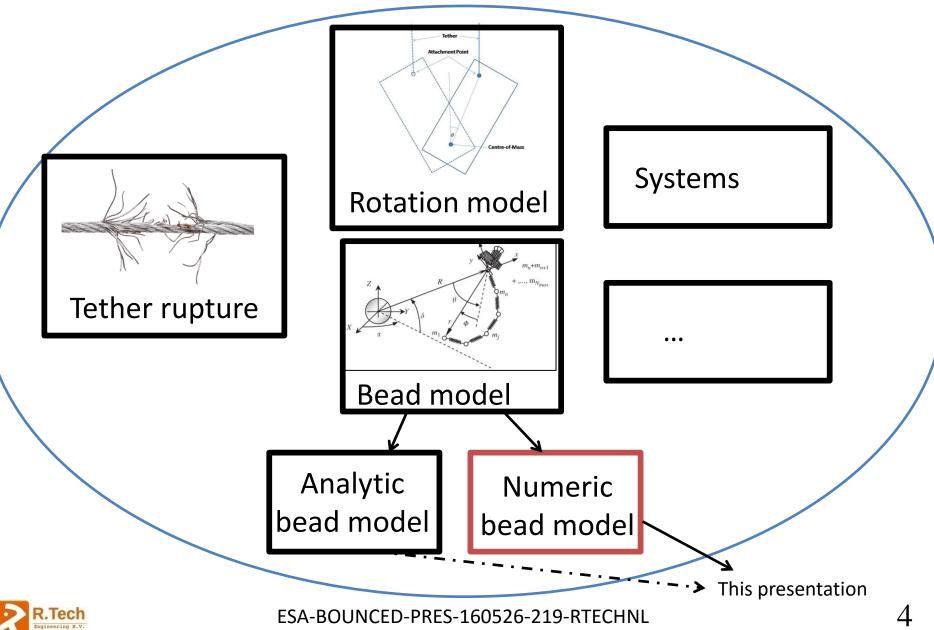
« BOUNCED » STUDY

- BOdies Under Connected Elastic Dynamics
 - ESA GSP Activity
 - Clean Space Initiative
 - Support to Active Debris Removal Concepts
 - Harpoon/Net
 - Spacecraft Requires Towing using a Tether
 - What do we know about fundamental tether dynamics?
 - What happens if control is lost?
 - <u>Elastic</u> tether:
 - Interests

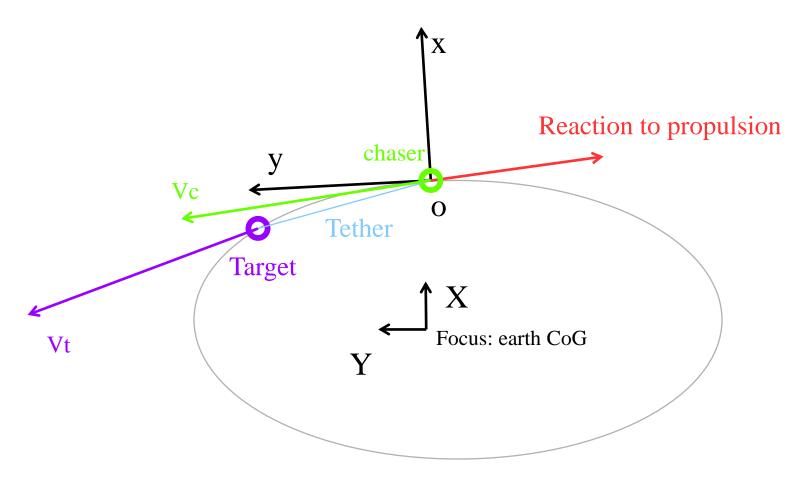




« BOUNCED »



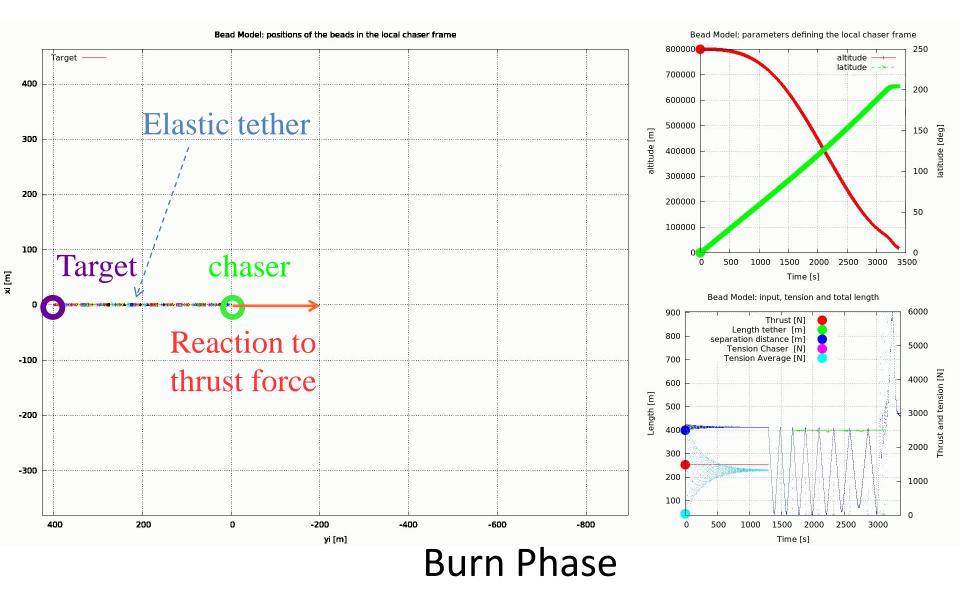
Description of problem: Target orbiting around the earth to be demised



Different regimes covered by BOUNCED

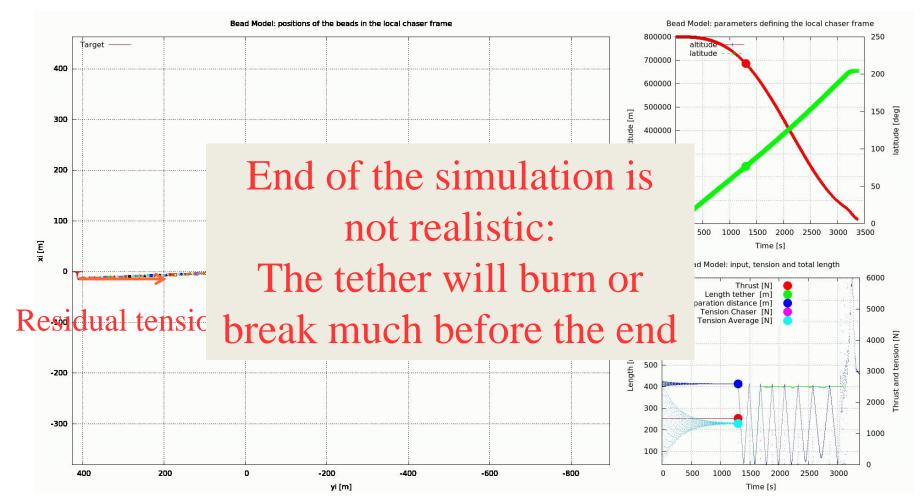


What are we modeling ? (1/2)





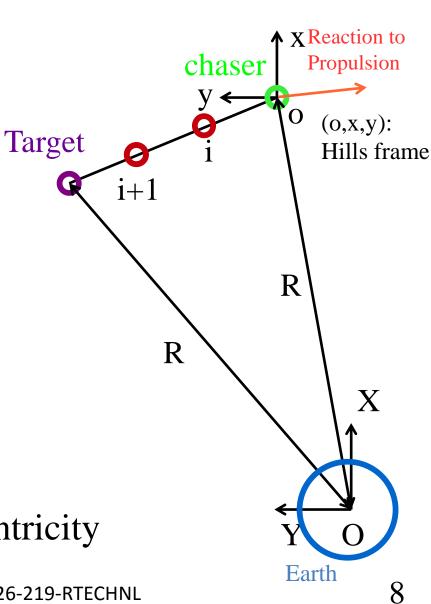
What are we modeling ? (2/2)



Relative orbital motion, retention & early atmosphere interaction

Bead Model: external forces and propulsion

- N beads:
 - • (1) Chaser
 - **O** (1) Target
 - \circ (N 2) Elastic tether
- External forces:
 - Gravitation
 - Inertia
 - Simple aerodynamics
- Propulsion force
- 2D effects and orbital eccentricity

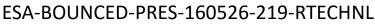


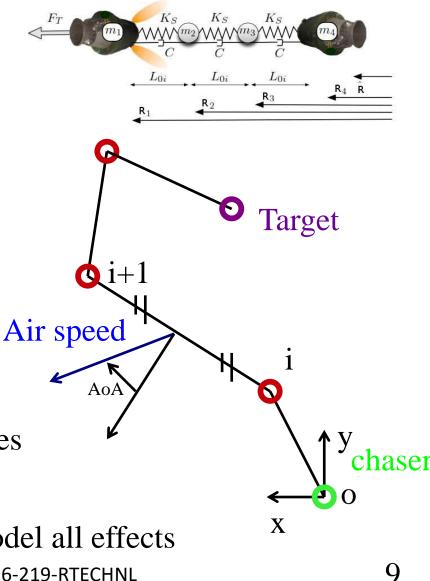


Bead Model: tether modeling

- Beads connection modeling:
 - Spring
 - Damper
- Parameters of the tether:
 - E : Young module
 - L : Length
 - Ks : Elasticity
 - C : Damping
 - mⁱ : Bead masses
 - N beads => Mods/frequencies
 - Drag coefficients

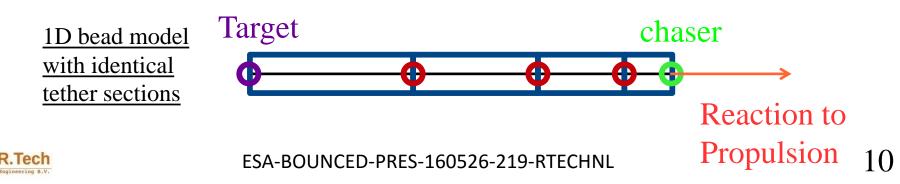
=> Numerical bead model: can model all effects





Analytical model

- Combination of 2 analytical models
- Main differences with numeric model:
 - Burn and retention phases:
 - 1D bead model
 - No external forces
 - Orbital phases:
 - No force apply on the tether beads (target and chaser only).
 - Low eccentricity approximation



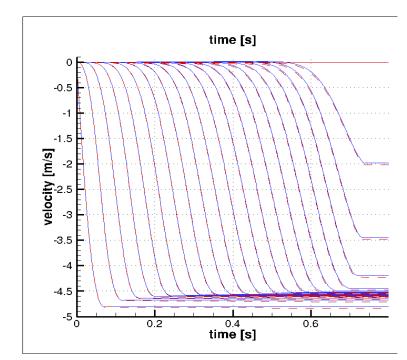
Resolution

- Analytic model:
 - Combination of 2 analytical models:
 - Tension phases: symmetric tridiagonal system
 - Orbital phases: Hills's equations
- Numeric model:
 - Implemented in Java
 - Equations solved by a numerical integrator



Validation

- Numeric model validated using:
 - Analytic model
 - Experimental data
- Cross validation between
 - Analytic model
 - Numeric model
- Different parameters
- ⇒ What have we learned from the cross comparison (parameter study) ?
- ⇒ What can be done with the analytic model ?



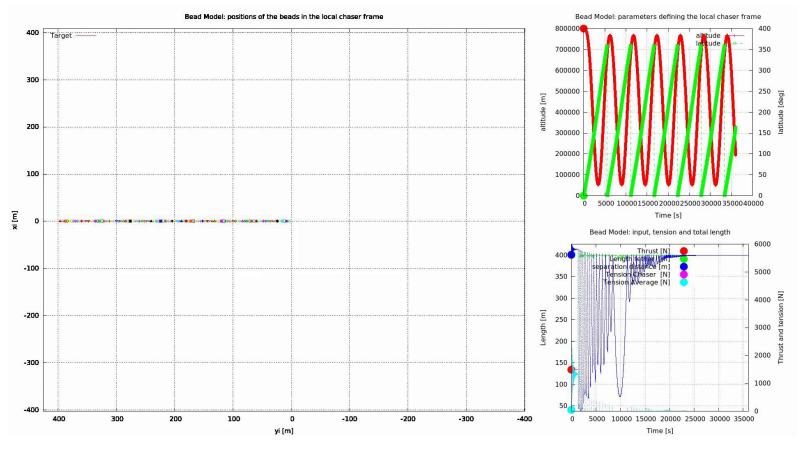
Example of comparison:

Velocities of the beads after breaking of the tether: 1D model (red) vs 2D at 800km (blue)



Results: Description of the motion without atmospheric drag (e.g. to mimic propulsion failure)

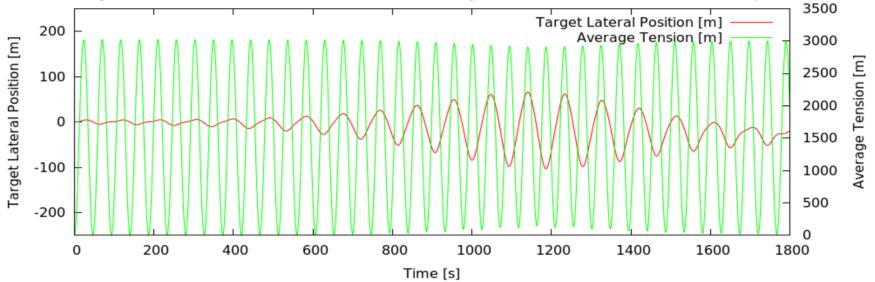
- Coriolis forces are dominant $=> 1^{st}$ pass misses suggests others miss
- System evolves to a gravity gradient stabilized oscillating state





Results: 2D lateral motion characterization - Burn phase

- Lateral 2D motion observed in numerical model
- The lateral motion is due to thrust not exactly in orbit direction.
- Frequency can be approximated by a pendulum analogy
- Minimum acceptable frequency can be predicted for the system
 - Analytic formula validated through numerical experiment.
- Limiting case shown: lateral and longitudinal oscillations synchronize

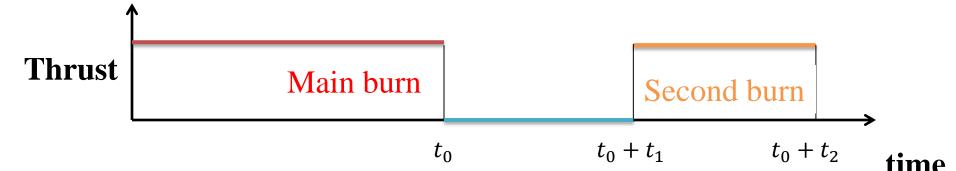


• Pretension and tether damping have limited impact on mitigation



Results: Input shaping to mitigate lose of control

- **Stored elastic energy => Post burn relative velocity** significant
- A second burn (Bang-off-bang) is used to mitigate the velocity
- <u>Objective</u>: null relative velocity at the end of the second burn

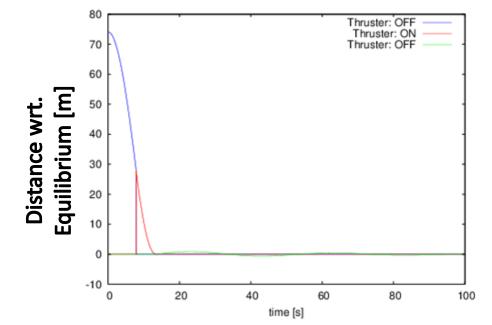


- Exact solution for undamped system (C=0) :
 - **Derived from the 1D analytical model** assuming:
 - End of main burn: Tether system is **fully damped**
- Solution for damped system (C!=0):
 - Based on exact solution for undamped system
 - Corrected with a term taking into account the damping (C)



Results: Input shaping to mitigate lose of control

- Solution with damping gives a good solution
- Numerical simulation validates the analytical solution:
 - Simulation starts at the end of the main burn (time=0)

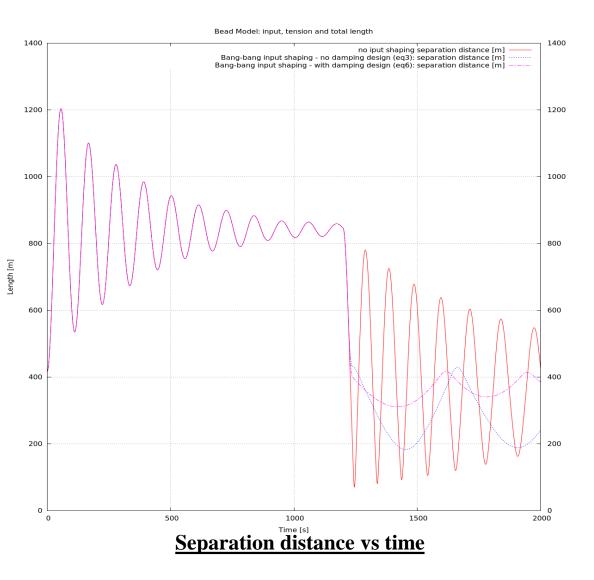


10% damping ratio system: Bang-off-bang using the damped scheme

NB: Solution (in green) are not representative of the system as soon as the distance is negative.



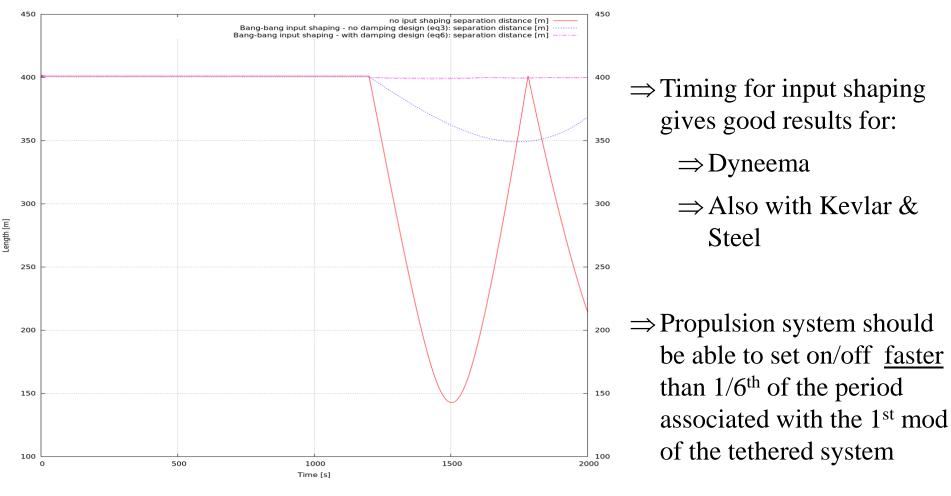
Results: Input shaping applied to Rubber



- Oscillations not fully damped at the end of the main burn
- => Timing for input shaping not appropriate



Results: Input shaping applied to Dyneema



Separation distance vs time



Modeling capabilities for elastic tether dynamics

Modeling	Analytic model	Numeric model
Retention phases	Error introduced by relative velocity mirrored after retention	Modeled, but rotation model also is an important aspect.
Slacking during burn phase	Not needed if enough pretension (50 N is OK) or damping	Snatch loads modeled when part of tether are slacked
Propellant mass	Not needed if oscillations damped at the end of burn	Time dependent propellant mass loss modeled
Lateral 2D motion	Not needed if tether not too elastic	Predict oscillations. Validate analytical formula for prediction of large lateral oscillations
Orbit eccentricity	Low eccentricity. Do not capture orbital dynamics precisely. But relatively good approximation for a few retention cycles	Modeled.
Multiple burn	No, but theory used to define times: off- on-off	Use to simulate mitigation of relative velocity at the end of burn
Multiple material tether	Limited: dashpot constant over stiffness ratio identical for each tether section	Difficult: time consuming
Multiple frequencies captured	Many beads: fast (seconds)	Many beads: slow (hours)



Thank you for your attention



Back up slides

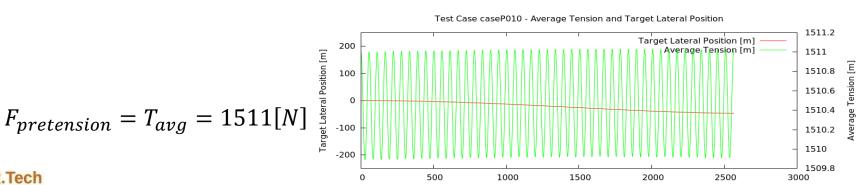
Baseline data

Quantity	Value	Unit
Mass target	8000	[kg]
Mass chaser	1000	[kg]
Ν	2	[-]
rho	5000	[kg/m^3]
E	1e9	[Pa]
Α	5.e-6	[m^2]
L	400	[m]
pretension	0	[N]
Burn Time	1200	[s]
Thrust Force	1700	[N]
Altitude	800	[km]
VX	0	[m/s]

Results: 2D lateral motion, pretension effect

>Lateral motion depends on: $|F_{pretension} - T_{avg}|$ >Average tether tension: $T_{avg} = F \frac{m_t}{m_c + m_t}$ >Impact negligible with a pretension of 50 [N]





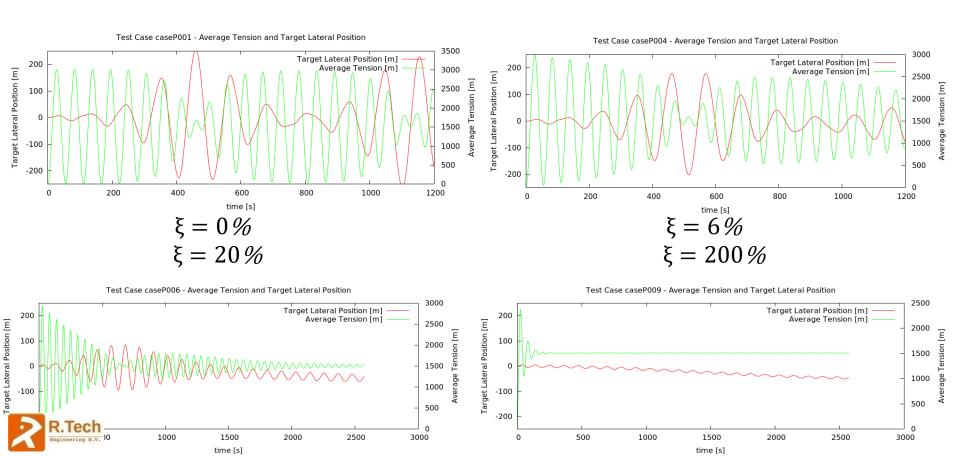
Time [s]

 $F_{pretension} = 0[N]$

R.Tech Engineering B.V.

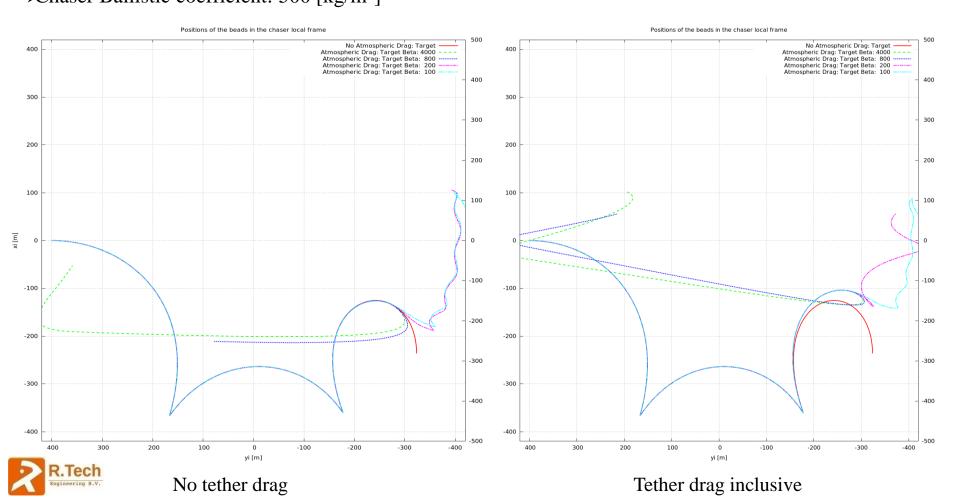
Results: 2D lateral motion, damping effect

Lateral/longitudinal coupling not altered by damping
 Longitudinal oscillations damped → Lateral oscillations damped until a certain point
 Simulations carried out with "p"=2:



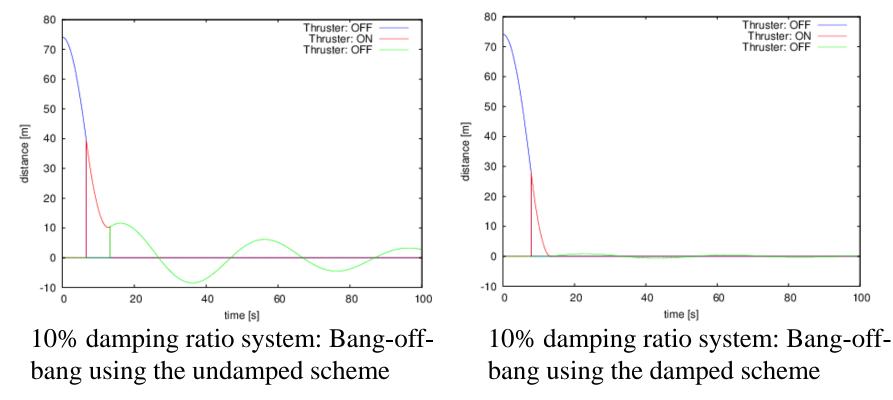
Atmosphere drag: low post burn velocity

Post burn relative velocity: -0.5 [m/s] Chaser Ballistic coefficient: 500 [kg/m²]



Results: Input shaping to mitigate lose of control

- Analytical solution <u>with damping</u> gives a good approximation
- Numerical simulation validate the analytical solution:
 - Simulation starts at the end of the main burn (time=0)



NB: Solution (in green) are not representative of the system as soon as the distance is negative.





Energies: definitions

Potential of gravitation:

$$V_{g} = \sum_{i=0}^{N-1} -m_{i} \frac{GM_{Earth}}{R_{i}}$$
Elasticity energy:

$$V_{e} = \frac{1}{2} \sum_{i=0}^{N-1} k_{i} (|\mathbf{R}_{i+1} - \mathbf{R}_{i}| - L_{i}^{eq})^{2}$$
Kinetic energy:

$$K_{T} = \frac{1}{2} \sum_{i=0}^{N-1} m_{i} |\mathbf{R}_{i+1} - \mathbf{R}_{i}|^{2}$$
The total energy:

$$U = U_{int} + U_{coG}$$
The total energy:

$$U = K_{T} + V_{g} + V_{e}$$
"Internal energy":

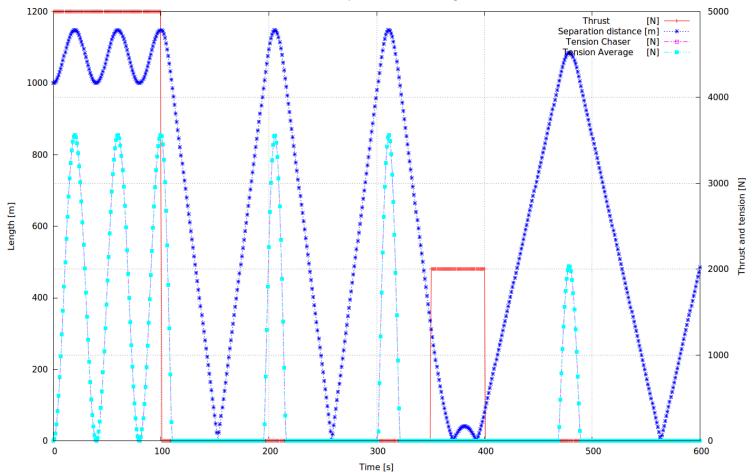
$$U_{int} = K_{int} + V_{e}$$
Kinetic energy
Center of gravity:

$$K_{coG} = \frac{1}{2} \frac{(\sum_{i=0}^{N-1} m_{i} \mathbf{R}_{i})^{2}}{\sum_{i=0}^{N-1} m_{i}}$$
"Energy of the center of
gravity":

$$U_{coG} = K_{coG} + V_{g}$$
Difference of Kinetic

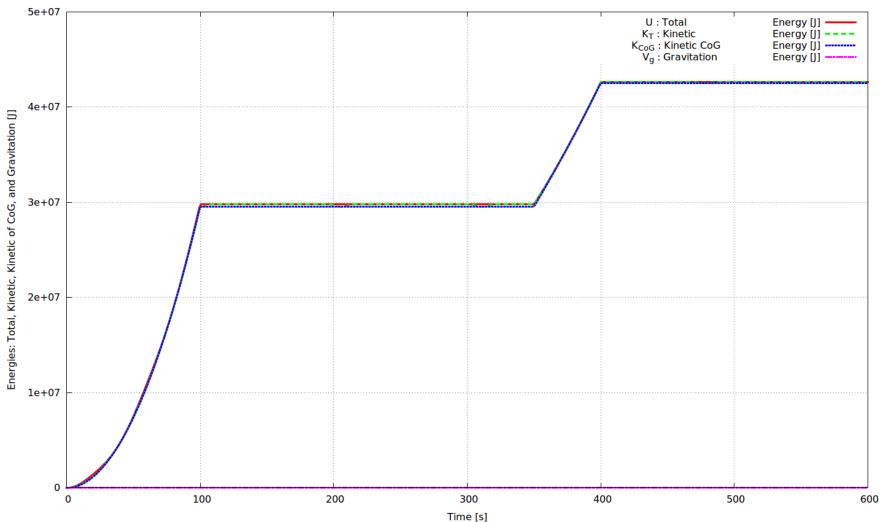
$$K_{int} = K_{T} - K_{coG}$$

Energies: 1D example



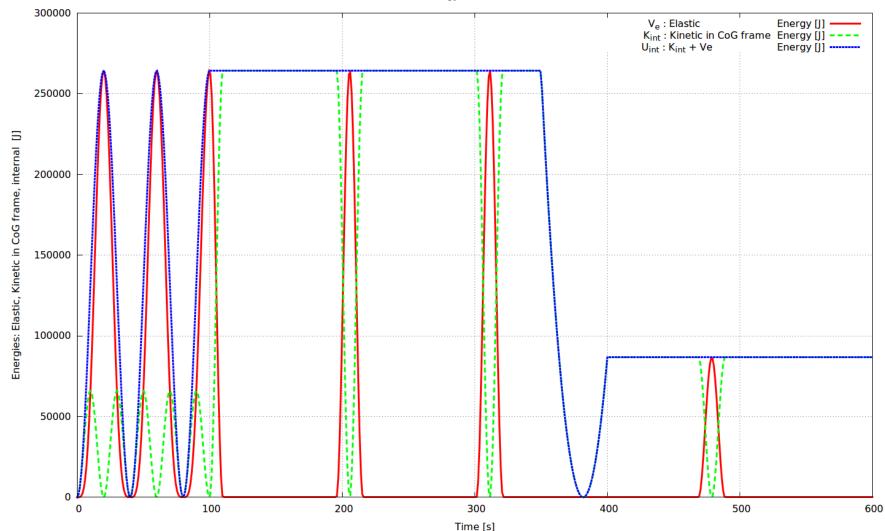
Bead Model: input, tension and total length

Energies:



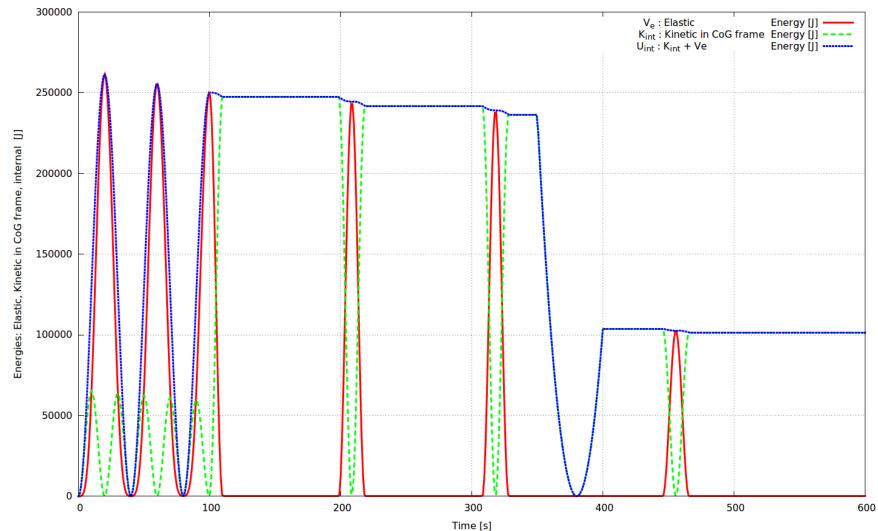
Energy contributions

Energies: without damping



Energy contributions

Energies: with damping



Energy contributions